

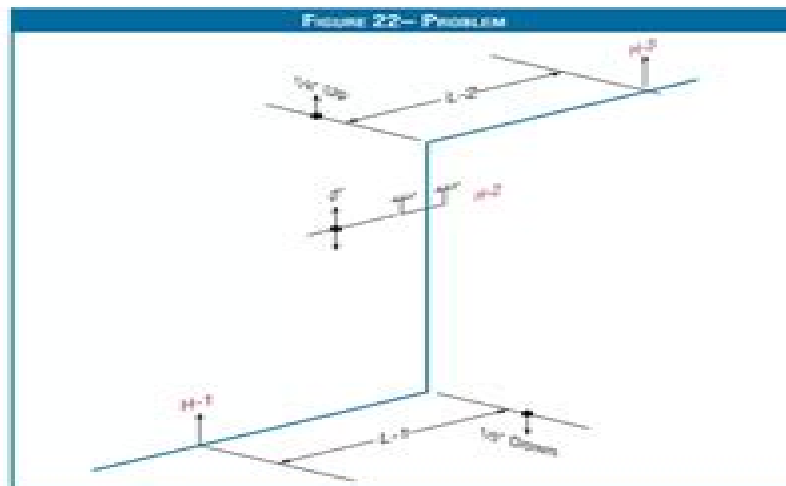
The amount of vertical movement at hanger H-2 will be proportional to its distance between H-3 and the riser, and can be approximated as shown in Figure 21:



Thus, H-2 would be selected as a variable spring hanger for .43 in. of downward vertical movement, and H-3 would be designed as a rigid hanger.

In the above problem the hanger locations were fixed. If this were not the case, and the hangers could be placed at any convenient location subject to usual hanger span limits, then H-2 would be placed at any distance 17.5 ft. or more from the riser. This would satisfy the condition that a maximum bending stress of 10,000 psi would result from the restraining effect of the hanger. If the allowable effect was given as a higher stress, then the hanger could be placed closer to the riser; if lower, the nearest rigid hanger would be placed a greater distance from the riser.

If the hanger were located closer to the riser, a greater restraining force would be applied to the pipe by the hanger. As the location is changed to a greater distance from the riser, a lesser force is required. As illustrated in the following sample problem, this force can be an important factor in the design load of the hanger.



PROBLEM

Given 10 in. Sch. 40 pipe, and allowable bending stress of 10,000 psi produced by the restraining effect of the hangers, Find:

- (1) L-1 and L-2 the distances to the nearest rigid hangers H-1 and H-3, see Figure 22.
- (2) The forces which the hangers must apply to the pipe to allow the $\frac{1}{4}$ in. and $\frac{1}{2}$ in. deflections resulting from the thermal expansion of the vertical pipe.

Solution:

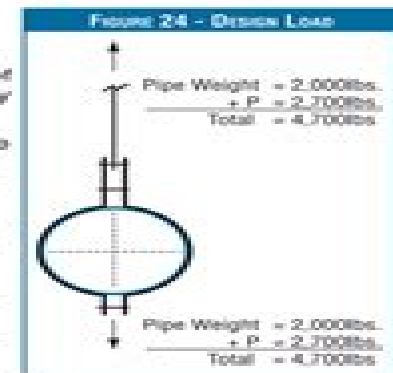
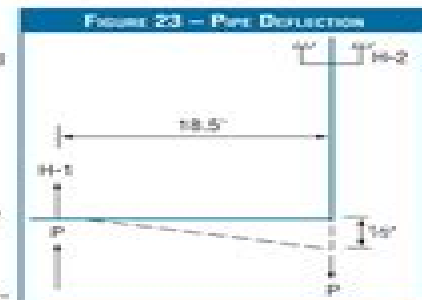
From the Chart on page 67 using values of $\frac{1}{4}$ in. deflection and 10 in. pipe, read L-1, as 18.5 ft., the distance from the riser to the rigid hanger H-1. Thus, at a distance of 18.5 ft., the hanger will exert sufficient force to deflect the pipe $\frac{1}{4}$ in., producing 10,000 psi bending stress. (See Fig. 23).

Use the Chart on page 69 to find the value of force P. For a pipe size of 10 in. and a span of 18.5 ft., read P as approximately 2,700 lb.

This force is applied by the pipe hanger H-1, and, therefore, must be included in the design load for H-1. In this instance, where the piping movement is in the downward direction, the force P is added to the pipe weight to be supported by Hanger H-1. If the pipe weight for H-1 were calculated as 2,000 lb., then the design load for the hanger components is 2,000 lb. + 2,700 lb., or 4,700 lb., as shown in Figure 24.

To solve for L-2 refer to the Chart on page 67 and, using values of $\frac{1}{2}$ in. deflection and 10 in. pipe, read L-2 as 13 ft., the distance to the proposed rigid hanger H-3. As discussed for H-1 of this problem, hanger H-3 must apply sufficient force to restrain the pipe vertically against the force resulting from the thermal expansion of the vertical piping above H-2.

The force P which is required at H-3 can be determined from the Chart on page 69. Using values for 10 in. pipe and a 13 ft. span, P is approximately 3,600 lb. Since this force restrains the upward movement of the pipe, it should be checked against the pipe weight load to assure that the hanger assembly can exert a force equal to the difference of the force P and the pipe weight load.



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JL Elias

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Table of Contents Itt Grinnell Pipe Support Design And Engineering

1. Understanding the eBook Itt Grinnell Pipe Support Design And Engineering
 - The Rise of Digital Reading Itt Grinnell Pipe Support Design And Engineering
 - Advantages of eBooks Over Traditional Books
2. Identifying Itt Grinnell Pipe Support Design And Engineering
 - Exploring Different Genres
 - Considering Fiction vs. Non-Fiction
 - Determining Your Reading Goals
3. Choosing the Right eBook Platform
 - Popular eBook Platforms
 - Features to Look for in an Itt Grinnell Pipe Support Design And Engineering
 - User-Friendly Interface
4. Exploring eBook Recommendations from Itt Grinnell Pipe Support Design And Engineering
 - Personalized Recommendations
 - Itt Grinnell Pipe Support Design And Engineering User Reviews and Ratings
 - Itt Grinnell Pipe Support Design And Engineering and Bestseller Lists

5. Accessing Itt Grinnell Pipe Support Design And Engineering Free and Paid eBooks
 - Itt Grinnell Pipe Support Design And Engineering Public Domain eBooks
 - Itt Grinnell Pipe Support Design And Engineering eBook Subscription Services
 - Itt Grinnell Pipe Support Design And Engineering Budget-Friendly Options
6. Navigating Itt Grinnell Pipe Support Design And Engineering eBook Formats
 - ePub, PDF, MOBI, and More
 - Itt Grinnell Pipe Support Design And Engineering Compatibility with Devices
 - Itt Grinnell Pipe Support Design And Engineering Enhanced eBook Features
7. Enhancing Your Reading Experience
 - Adjustable Fonts and Text Sizes of Itt Grinnell Pipe Support Design And Engineering
 - Highlighting and Note-Taking Itt Grinnell Pipe Support Design And Engineering
 - Interactive Elements Itt Grinnell Pipe Support Design And Engineering
8. Staying Engaged with Itt Grinnell Pipe Support Design And Engineering
 - Joining Online Reading Communities
 - Participating in Virtual Book Clubs
 - Following Authors and Publishers Itt Grinnell Pipe Support Design And Engineering
9. Balancing eBooks and Physical Books Itt Grinnell Pipe Support Design And Engineering
 - Benefits of a Digital Library
 - Creating a Diverse Reading Collection Itt Grinnell Pipe Support Design And Engineering
10. Overcoming Reading Challenges
 - Dealing with Digital Eye Strain
 - Minimizing Distractions
 - Managing Screen Time
11. Cultivating a Reading Routine Itt Grinnell Pipe Support Design And Engineering
 - Setting Reading Goals Itt Grinnell Pipe Support Design And Engineering
 - Carving Out Dedicated Reading Time
12. Sourcing Reliable Information of Itt Grinnell Pipe Support Design And Engineering
 - Fact-Checking eBook Content of Itt Grinnell Pipe Support Design And Engineering
 - Distinguishing Credible Sources
13. Promoting Lifelong Learning

- Utilizing eBooks for Skill Development
- Exploring Educational eBooks

14. Embracing eBook Trends

- Integration of Multimedia Elements
- Interactive and Gamified eBooks

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