

1. (a) Explain intensive and extensive properties with examples. 4
- (b) Give reasons why the Carnot cycle can not be considered as the practical power plant cycle even-though it's efficiency is maximum. 4
- (c) An inventor claims that a new heat cycle will develop 400 W for a heat supply of 35.5 kJ/min. Is his claim possible ? Given that source is at 200 K and sink is at 850 K. 4
- (d) 2 kg of steam is at 8 bar and 0.8 dry. Determine it's enthalpy and volume. 4
- (e) What is Cut-off ratio ? How does it affect the thermal efficiency of Diesel cycle ? 4

2. (a) Air at a temperature of 20 °C passes through a heat exchanger at a velocity of 40 m/s where it's temperature is raised to 820 °C. It then enters a turbine with same velocity of 40 m/s and expands till the temperature falls to 620 °C. On leaving the turbine, the air is taken at a velocity of 55 m/s to a nozzle where it expands until the temperature has fallen to 510 °C. If the air flow rate is 2.5 kg/s, calculate : 10
 - (i) rate of heat transfer to the air in the heat exchanger.
 - (ii) the power output from the turbine assuming no heat loss.
 - (iii) the velocity at exit from the nozzle, assuming no heat loss.
- (b) Define : 5
 - (i) Availability
 - (ii) Dead state
 - (iii) Irreversibility
 - (iv) Effectiveness.
- (c) For the same compression ratio and heat supplied compare Otto, Diesel and Dual cycle with the help of P-V and T-S diagram. 4

3. (a) State and prove Carnot theorem. 7
- (b) A Rankine cycle operates between the pressure 15 bar and 0.01 bar. The initial degree of superheat is 100 °C. Assuming isentropic efficiency of expansion 85%, calculate : 8
 - (i) Pump work
 - (ii) Actual turbine work
 - (iii) Thermal efficiency
- (c) What is Joule-Thompson coefficient ? What conclusions can be drawn from a given values of this coefficient ? 4

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